[34] Venkataramana, J., Degaleesan, T.E., Laddha, G.S., Can. J. Chem. Eng 58 (1980) pp. 206-211.
[35] Zhu, S., Zhang, B., Shen, Z., Wang, J., Chem. Ind. Eng (China) (1982) No. 1, p. 1.
[36] Kamath, M.S., Subba Rau, M.G., Can. J. Chem. Eng 63 (1985) pp. 578-584
[37] Coulaloglou. C.A., Tavlarides, L.L., Chem. Eng Sci. 32 (1977) pp. 1289-1297.
[38] Eid, K., Thesis, L’Institut National Polytechnique de Toulouse,

France, 1984.
[39] Kumar, A., Hartland, S., Trans. Inst. Chem. Eng 60 (1982) pp. 35-39.
[40] Mugele, R.A., Evans, H.D., Ind. Eng Chem. 43 (1951) No. 6, pp. 1317-1324.
[41] Slater, M.J., Unpublished work, 1989.
[42] Grace, J.R., Wairegi, T., Nguyen, T.H., Trans. Inst. Chem. Eng 54 (1976) pp. 167-174.

# Modelling of Pressure Drop in Packed Columns 

Reinhard Billet and Michael Schultes*

The correct choice of packing is of decisive importance for optimum process efficiency in the operation of two-phase countercurrent columns. An important criterion for this choice is the pressure drop in the gas flow. Theoretical relationships are derived for calculating the pressure drop in beds with dry and trickle packings. It has been demonstrated by comprehensive experiments that these relationships allow the pressure drop to be determined more accurately than by previous methods. The experiments were performed at the Department of Thermal Separation Processes of Bochum University on 54 different packed beds, using 24 different systems.

## 1 Introduction

In the past few years, highly effective new packings have been developed for absorption, desorption and rectification columns, used for the separation of liquid and gas mixtures in the process industries. They feature minimum maldistribution, large interfacial areas and reduced pressure drops in the gas phase.

The differences between the geometry of the new types of packing and that of conventional designs necessitated a critical check of the existing design rules for calculating the pressure drop. Most equations found in the literature for its determination in absorption, desorption and rectification columns are empirical or semi-empirical. They have been derived largely from studies on conventional packings and cannot be unreservedly applied to modern designs. The aim of this paper is to develop a new theoretical model, valid for both conventional and modern types of packing, for the calculation of pressure drop in two-phase countercurrent columns.

* Prof. Dr.-Ing. R. Billet and Dr.-Ing. M. Schultes, Ruhr-Universität Bochum. Department of Thermal Separation Processes, D-4630 Bochum, Universitätsstraße 150.

In order to describe the flow in a packed trickle bed, it is assumed that the bed is equivalent to a multiplicity of flow channels through which the liquid of density ${ }^{1)} \varrho_{\mathrm{L}}$ and viscosity $\eta_{\mathrm{L}}$ flows downwards as a film of thickness $\mathrm{s}_{\mathrm{o}}$ at a local velocity $\bar{u}_{\mathrm{L}, \mathrm{s}}$. If inertia forces are neglected, gravity and shear forces in the laminar-flow film, Eq. (1), are held in equilibrium with the frictional forces by the shear stress $\tau_{V}$ in the vapour at the surface of the film, Eq. (2). In Eq. (2), $\varrho_{V}$ is the gas density, $\bar{u}_{V}$ the average effective gas velocity and $\psi$ the resistance coefficient for gas flow [3]:
$\frac{\mathrm{d}\left(\eta_{\mathrm{L}} \frac{\mathrm{d} \bar{u}_{\mathrm{L}, \mathrm{S}}}{\mathrm{d} s}\right)}{\mathrm{d} s}=-\varrho_{\mathrm{L}} g$,
$\tau_{\mathrm{v}}=-\psi \frac{\varrho_{\mathrm{V}} \bar{u}_{\mathrm{V}}^{2}}{2}$.

[^0]
## 2 Pressure Drop in a Gas Stream Flowing through Dry Beds

The pressure drop $\Delta p_{o} / H$ of a stream of gas flowing through dry bed of height $H$ can be determined from the shear force/pressure equilibrium and the Newtonian friction law. Eq. (3), in which $d_{\mathrm{h}}$ is the hydraulic diameter and $\psi_{\mathrm{o}}$ the resistance coefficient for the dry bed, can thus be derived from Eq. (2):
$\frac{\Delta p_{\mathrm{o}}}{H}=\psi_{\mathrm{o}} \frac{4}{d_{\mathrm{h}}} \frac{\varrho_{\mathrm{v}} \bar{u}_{\mathrm{V}}^{2}}{2}$.
The average effective gas velocity can be calculated as the ratio of the average gas velocity per unit cross-sectional area of column $u_{\mathrm{v}}$ to the void fraction $\varepsilon$, Eq. (4). The product of gas velocity and the square root of gas density yields the gas capacity factor $F_{\mathrm{V}}$, Eq. (5). The hydraulic diameter $d_{\mathrm{h}}$ is fixed by the free bed volume ( $V_{\mathrm{S}}-V_{\mathrm{P}}$ ), where $V_{\mathrm{S}}$ is the column volume, $V_{\mathrm{P}}$ the bed volume and $A_{\mathrm{P}}$ the total area of packing. It can thus be described by the void fraction $\varepsilon$ and the geometric area $a$ per unit volume of packing, Eq. (6)
$\bar{u}_{\mathrm{V}}=\frac{u_{\mathrm{V}}}{\varepsilon}$,
$F_{\mathrm{V}}=u_{\mathrm{V}} e_{\mathrm{V}}^{1 / 2}$,
$d_{\mathrm{h}}=4 \frac{V_{\mathrm{S}}-V_{\mathrm{P}}}{A_{\mathrm{P}}}=4 \frac{\varepsilon}{a}$.
Inserting Eqs (4) - (6) into Eq. (3) yields the following expression for the pressure drop per unit height, Eq. (7)
$\frac{\Delta p_{\mathrm{o}}}{H}=\psi_{\mathrm{o}} \frac{a}{\varepsilon^{3}} \frac{F_{v}^{2}}{2}$.
In a real packed bed, the local void fraction differs from the theoretical value $\varepsilon$, depending on the column diameter $d_{\text {s }}$ because there is more free space at the wall of the column. The difference can be accounted for by a wall factor $K$, Eq. (8). This gives Eq. (9) [6]
$\frac{1}{K}=1+\frac{2}{3} \frac{1}{1-\varepsilon} \frac{d_{\mathrm{P}}}{d_{\mathrm{S}}}$,
$\frac{\Delta p_{\mathrm{o}}}{H}=\Psi_{\mathrm{o}} \frac{a}{\varepsilon^{3}} \frac{F_{\mathrm{v}}^{2}}{2} \frac{1}{K}$.

The resistance coefficient $\psi_{\mathrm{o}}$ in Eq. (9) must be determined empirically. Thus, Eq. (10) is based on the data obtained in experimental studies on 54 different types of packings. It includes the effect exerted on the gas flow by the vapour Reynolds number $\operatorname{Re}_{\mathrm{v}}$, defined by Eq. (11), in which $\nu_{\mathrm{v}}$ is the kinematic viscosity of the gas and $d_{\mathrm{P}}$ the particle diameter. It is seen from Eq. (12) that the particle diameter depends on the ratio of the volume of packing $V_{\mathrm{P}}$ to its total area $A_{\mathrm{P}}$ and can thus be described by $\varepsilon$ and $a$.
$\psi_{\mathrm{o}}=C_{\mathrm{P}}\left(\frac{64}{\operatorname{Re}_{\mathrm{V}}}+\frac{1.8}{\operatorname{Re}_{\mathrm{V}}^{0.08}}\right)$,
$\operatorname{Re}_{\mathrm{V}}=\frac{u_{\mathrm{V}} d_{\mathrm{P}}}{(1-\varepsilon) \nu_{\mathrm{V}}} K$,
$\mathrm{d}_{\mathrm{P}}=6 \frac{V_{\mathrm{P}}}{A_{\mathrm{P}}}=6 \frac{1-\varepsilon}{a}$.
The term $\psi_{o}$ defined by Eq. (10) is known from literature and has been confirmed by the studies carried out at the Department of Thermal Separation Processes of Bochum University. The constant $C_{\mathrm{P}}$ characterizes the geometry and surface properties of dry packing and is therefore specific for a given type of packing. Its values are listed in Tables 1a and 1 b .

An example is presented in Fig. 1 which shows the pressure drop per unit height $\Delta p_{o} / H$ of a dry bed of 32 mm plastic Envipac rings as a function of the gas capacity factor $F_{\mathrm{v}}$. The associated relationship between the resistance coefficient $\psi_{o}$ and the gas Reynolds number $\mathrm{Re}_{\mathrm{v}}$ is shown in Fig. 2. At low loads, the downward slope of the curve shown in Fig. 2 becomes steeper. In this range, up to Reynolds numbers of $\mathrm{Re}_{\mathrm{v}}$ $\approx 2100$, the gas flow is laminar, and the first summand in Eq. (10) governs the shape of the curve. Above this value, the flow becomes turbulent, and the second summand in Eq. (10) becomes the most decisive.

## 3 Pressure Drop in Gas Flows through Packed Trickle Beds

If the packing is wetted by a liquid, the column volume available for gas flow becomes reduced by the volume of the liquid fraction. Hence, if the volume of liquid in the bed $V_{L}$ is expressed as a fraction of the column volume $V_{\mathrm{S}}$, the liquid holdup $h_{\mathrm{L}}$ can be described by Eq. (13), and the effective void fraction $\varepsilon_{\mathrm{e}}$ by Eq. (14).
$h_{\mathrm{L}}=\frac{V_{\mathrm{L}}}{V_{\mathrm{S}}}$
$\varepsilon_{\mathrm{e}}=\left(\varepsilon-h_{\mathrm{L}}\right)$
Eq. (15) is then obtained by substituting $\varepsilon_{\mathrm{e}}=\varepsilon-h_{\mathrm{L}}$ for $\varepsilon$ in Eq. (9) and by introducing a wetting factor $f_{\mathrm{w}}$ which reflects the change in the packing surface area as a result of wetting. It is thus valid for determining the pressure drop per unit height in a packed trickle bed with a resistance coefficient for two-phase flow $\psi_{\mathbf{L}}$ :
$\frac{\Delta p}{H}=\psi_{\mathrm{L}} \frac{f_{\mathrm{w}} a}{\left(\varepsilon-h_{\mathrm{L}}\right)^{3}} \frac{F_{\mathrm{v}}^{2}}{2} \frac{1}{K}$.
The excess pressure drop of the gas stream in the trickle bed over that in the dry bed corresponds to their ratio $\Delta p / \Delta p_{o}$, as expressed by Eq. (16)
$\frac{\Delta p}{\Delta p_{\mathrm{o}}}=\frac{\psi_{\mathrm{L}}}{\psi_{\mathrm{o}}} f_{\mathrm{w}}\left(\frac{\varepsilon}{\varepsilon-h_{\mathrm{L}}}\right)^{3}$,
$f\left(h_{\mathrm{L}}\right)=\left(\frac{\varepsilon}{\varepsilon-h_{\mathbf{L}}}\right)^{3}$.

Table 1a. Characteristic data and constants $C_{\mathrm{P}}$ of Eqs (10) and (21) for dumped packings.

| Dumped packings | Material | Size | $\begin{gathered} N \\ {\left[1 / \mathrm{m}^{3}\right]} \end{gathered}$ | $\begin{gathered} a \\ {\left[\mathrm{~m}^{2} / \mathrm{m}^{3}\right]} \end{gathered}$ | $\begin{gathered} \varepsilon \\ {\left[\mathrm{m}^{3} / \mathrm{m}^{3}\right]} \end{gathered}$ | $C_{\text {P }}$ |
| :---: | :---: | :---: | :---: | :---: | :---: | :---: |
| Pall rings | Metal | 50 mm | 6242 | 112.6 | 0.951 | 0.763 |
|  |  | 38 mm | 15772 | 149.6 | 0.952 | 1.003 |
|  |  | 35 mm | 19517 | 139.4 | 0.965 | 0.967 |
|  |  | 25 mm | 53900 | 223.5 | 0.954 | 0.957 |
|  |  | 15 mm | 229225 | 368.4 | 0.933 | 0.990 |
|  | Plastic | 50 mm | 6765 | 111.1 | 0.919 | 0.698 |
|  |  | 35 mm | 17000 | 151.1 | 0.906 | 0.927 |
|  |  | 25 mm | 52300 | 225.0 | 0.887 | 0.865 |
|  | Ceramic | 50 mm | 6215 | 116.5 | 0.783 | 0.662 |
| Ralu rings | Plastic | 50 mm | 5770 | 95.2 | 0.938 | 0.468 |
|  |  | 50 mm , hydr. | 5720 | 94.3 | 0.939 | 0.439 |
| Hiflow rings | Metal | 50 mm | 5000 | 92.3 | 0.977 | 0.421 |
|  |  | 25 mm | 40790 | 202.9 | 0.962 | 0.689 |
|  | Plastic | 90 mm | 1340 | 69.7 | 0.968 | 0.276 |
|  |  | 50 mm | 6815 | 117.1 | 0.925 | 0.327 |
|  |  | 50 mm , hydr. | 6890 | 118.4 | 0.925 | 0.311 |
|  |  | 25 mm | 46100 | 194.5 | 0.918 | 0.741 |
|  | Ceramic | 75 mm | 1904 | 54.1 | 0.868 | 0.435 |
|  |  | 50 mm | 5120 | 89.7 | 0.809 | 0.538 |
|  |  | 35 mm | 16840 | 108.3 | 0.833 | 0.621 |
|  |  | $20 \mathrm{~mm}, 4$ webs | 121314 | 286.2 | 0.758 | 0.628 |
| Hiflow rings Super | Plastic | 50 mm | 6050 | 82.0 | 0.942 | 0.414 |
| NOR PAC rings | Plastic | 50 mm | 7330 | 86.8 | 0.947 | 0.350 |
|  |  | 35 mm | 17450 | 141.8 | 0.944 | 0.371 |
|  |  | 25 mm,type B | 47837 | 193.5 | 0.921 | 0.397 |
|  |  | $25 \mathrm{~mm}, 10$ webs | 44346 | 179.4 | 0.927 | 0.383 |
|  |  | 22 mm | 69274 | 249.0 | 0.913 | 0.397 |
|  |  | 15 mm | 193738 | 311.4 | 0.918 | 0.365 |
| Raflux rings | Plastic | 15 mm | 193522 | 307.9 | 0.894 | 0.595 |
| VSP rings | Metal | $50 \mathrm{~mm}, \mathrm{no} .2$ | 7841 | 104.6 | 0.980 | 0.773 |
|  |  | 25 mm , no. 1 | 33434 | 199.6 | 0.975 | 0.782 |
| Envipac rings | Plastic | 80 mm, no. 3 | 2000 | 60.0 | 0.955 | 0.358 |
|  |  | 60 mm , no. 2 | 6800 | 98.4 | 0.961 | 0.338 |
|  |  | 32 mm , no. 1 | 53000 | 138.9 | 0.936 | 0.549 |
| Top-pak | Aluminium | 50 mm | 6947 | 106.6 | 0.956 | 0.604 |
| Bialecki rings | Metal | 50 mm | 6278 | 121.0 | 0.966 | 0.719 |
|  |  | 35 mm | 19303 | 164.4 | 0.965 | 1.011 |
|  |  | 25 mm | 55000 | 238.0 | 0.940 | 0.891 |
| Raschig rings | Ceramic | 25 mm | 48175 | 185.4 | 0.662 | 1.329 |
| Intalox saddles | Plastic | 50 mm | 8656 | 122.1 | 0.908 | 0.758 |
|  | Ceramic | 50 mm | 8882 | 114.6 | 0.761 | 0.747 |
| Hiflow saddles | Plastic | 50 mm | 9939 | 86.4 | 0.938 | 0.454 |
| Tellerettes | Plastic | 25 mm | 35365 | 182.0 | 0.900 | 0.538 |
| Hackettes | Plastic | 45 mm | 12252 | 133.4 | 0.931 | 0.399 |

hydr. = hydrophilized.

Table 1b. Characteristic data and constants $C_{P}$ of Eqs (10) and (21) for regular packings.

| Regular packings | Material | Size | $N$ <br> $\left[1 / \mathrm{m}^{3}\right]$ | $a$ <br> $\left[\mathrm{~m}^{2} / \mathrm{m}^{3}\right]$ | $\varepsilon$ <br> $\left[\mathrm{m}^{3} / \mathrm{m}^{3}\right]$ |
| :--- | :--- | :--- | :--- | :--- | :--- | :--- |
| Pall rings | Ceramic | 50 mm | 7502 | 155.2 | 0.754 |
| Hiflow rings | Plastic | 50 mm | 7640 | 131.3 | 0.916 |
| Ralu pak | Metal | YC-250 | 8150 | 140.1 | 0.911 |

hydr. = hydrophilized


Fig. 1. Pressure drop of dry bed filled with 32 mm plastic Envipac rings as a function of gas capacity factor.


Gos Reynolds number $\mathrm{Re}_{\mathrm{v}}$
Fig. 2. Log-log plot of resistance factor for dry bed, packed with 32 mm plastic Envipac rings as a function of gas Reynolds number.

Eq. (16) indicates clearly that the increase in pressure drop can be described by the product of the ratio of resistance coefficients for two-phase and one-phase flow, the wetting factor $f_{\mathrm{w}}$ and an additional function $f\left(h_{\mathrm{L}}\right)$, Eq. (17), in which the liquid hold-up is a quantity which depends on the load.

Solution of Eqs (1) and (2) leads to Eq. (18), which describes the liquid hold-up over the entire loading range up to the flood point; and to Eq. (19), which describes the liquid hold-up to the loading point [ $3-5,9$ ]:
$h_{\mathrm{L}}=\left[\frac{a^{2} \eta_{\mathrm{L}} u_{\mathrm{L}}}{\frac{1}{3} g \varrho_{\mathrm{L}}-\frac{1}{4} \psi_{\mathrm{L}} \frac{a}{h_{\mathrm{L}}\left(\varepsilon-h_{\mathrm{L}}\right)^{2}} u_{\mathrm{V}}^{2} \varrho_{\mathrm{V}}}\right]^{1 / 3}$,
$h_{\mathrm{L}}=h_{\mathrm{L}, \mathrm{S}}=\left(\frac{12 \eta_{\mathrm{L}} u_{\mathrm{L}} a^{2}}{g \varrho_{\mathrm{L}}}\right)^{1 / 3}$.
Eq. (19) allows Eq. (17) to be solved and $f\left(h_{\mathrm{L}}\right)$ shown graphically as a function of the liquid load $u_{\mathrm{L}}$. This is demonstrated in Fig. 3, for which the 50 mm plastic Hiflow rings were taken as an example. As $u_{\mathrm{L}}$ decreases, the function $f\left(h_{\mathrm{L}}\right)$ approaches a limiting value of unity because the liquid hold-up tends to zero and thus its effect subsides. The higher the value selected for the liquid load, the greater is the effect exerted by the liquid in the column on the value of the function $f\left(h_{\mathrm{L}}\right)$ and the pressure drop will increase continuously.


Fig. 3. Experimental pressure drop ratios and values of hold-up function versus hiquid load for 50 mm plastic Hiflow rings.

Fig. 3 also shows the experimentally determined pressure drop ratios $\Delta p / \Delta p_{\mathrm{o}}$ as a function of the liquid load. It can be seen that the function $f\left(h_{L}\right)$ reproduces the experimental values fairly accurately. Hence, a new equation (Eq. (20)) can be written for the pressure drop ratio $\Delta p / \Delta p_{\mathrm{o}}$. As the liquid trickles through the bed, static hold-up occurs at the points of contact between the individual packings and in the interspaces, with the liquid forming a film on the surface of the packing. Hence, the surface structure differs from that during gas flow through a dry bed and this is reflected by the additional term $f(S)$ in Eq. (20):
$\frac{\Delta p}{\Delta p_{\mathrm{o}}}=f(S)\left(\frac{\varepsilon}{\varepsilon-h_{\mathrm{L}}}\right)^{x}$.
Combining Eqs (16) and (20) and substituting the right hand side of Eq. (10) for $\psi_{o}$ gives rise to Eq. (21) for the resistance coefficient $\psi_{\mathrm{L}}^{\prime}$ for two-phase flow:
$\psi_{\mathrm{L}}^{\prime}=\psi_{\mathrm{L}} f_{\mathrm{W}}=C_{\mathrm{P}} f(S)\left(\frac{64}{\operatorname{Re}_{\mathrm{V}}}+\frac{1.8}{\operatorname{Re}_{\mathrm{V}}^{0.08}}\right)\left(\frac{\varepsilon-h_{\mathrm{L}}}{\varepsilon}\right)^{(3-x)}$.

Eqs (15) and (21) were verified against the results obtained on a large number of packings. The verification included the effects exerted by the various physical properties of 24 different systems (see Table 2), including those intended for purely hydraulic studies and mixtures intended for absorption, desorption and rectification [1, 2]. Evaluation of thus obtained comprehensive data revealed that, in Eqs (20) and (21), the numerical value of exponent $x$ was 1.5 and that the expression $f(S)$ could be replaced by a function Eq. (22), which depends on the Reynolds number of liquid $\mathrm{Re}_{\mathrm{L}}$, Eq. (23). Values of the constant $C_{\mathrm{P}}$ in Eq. (21) are listed together with the characteristic data of the various types of packing in Tables 1a and 1 b .
$f(S)=\exp \left(\frac{\operatorname{Re}_{\mathrm{L}}}{200}\right) \quad$ for $x=1.5$,
$\operatorname{Re}_{\mathrm{L}}=\frac{u_{\mathrm{L}} \varrho_{\mathrm{L}}}{a \eta_{\mathrm{L}}}$.
Empirical values of the pressure drop $\Delta p / H$ in a trickle bed of 32 mm plastic Envipac rings are plotted against the gas capacity factor $F_{\mathrm{V}}$ in Fig. 4. Fig. 5 shows the corresponding values of the resistance coefficient $\psi_{\mathrm{L}}^{\prime}=\psi_{\mathrm{L}} f_{\mathrm{W}}$ as a function of the liquid load $u_{\mathrm{L}}$.

Mathematical prediction of the liquid hold-up $h_{\mathrm{L}}$ from Eq. (19) is restricted to the range extending up to the loading point. Fig. 6 shows the fundamental relationship between the liquid holdup $h_{\mathrm{L}}$ and the ratio of gas velocity $u_{\mathrm{V}}$ to that at the flood point $u_{\mathrm{V}, \mathrm{Fl}}$. Up to the loading point $S, h_{\mathrm{L}}$ is practically independent of gas velocity and is equal to $h_{\mathrm{L}, \mathrm{s}}$. Above this point, the shear forces, acting in the gas, support the liquid film until the liquid hold-up at the flood point attains the value of $h_{\mathrm{L}, \mathrm{FI}}$. These boundary conditions are described by Eqs (24) and (25).

1. Boundary condition: for $\frac{u_{\mathrm{V}}}{u_{\mathrm{V}, \mathrm{F} 1}}=0 \Rightarrow h_{\mathrm{L}}=h_{\mathrm{L}, \mathrm{S}}$.


Fig. 4. Pressure drop of trickle bed with 32 mm plastic Envipac rings as a function of gas capacity factor at various liquid loads.


Fig. 5. Resistance factor for trickle 32 mm plastic Envipac rings as a function of liquid load.


Fig. 6. Fundamental relationship between liquid hold-up and gas velocity for countercurrent flow columns.
2. Boundary condition: for $\frac{u_{\mathrm{V}}}{u_{\mathrm{V}, \mathrm{Fl}}}=1 \Rightarrow h_{\mathrm{L}}=h_{\mathrm{L}, \mathrm{Fl}}$.

Eq. (26) is an empirical equation which describes $h_{\mathrm{L}}$ in Fig. 6 and satisfies the boundary conditions
$h_{\mathrm{L}}=b+c\left(\frac{u_{\mathrm{V}}}{u_{\mathrm{V}, \mathrm{FI}}}\right)^{n}$.
Eq. (27) is then derived from Eqs (24) and (25), with an exponent $n=13$, obtained from experimental investigations; $h_{\mathrm{L}, \mathrm{S}}$ in this equation is the hold-up as defined by Eq. (19) and $h_{\mathrm{L}, \mathrm{Fl}}$ is the liquid hold-up at the flood point, as given by Eq. (28) [3]. The term in parentheses in Eq. (28) accounts for the effect of viscosity and density of the substance (subscript L) compared to the corresponding values for water at $20^{\circ} \mathrm{C}$ (subscript W ).
$h_{\mathrm{L}}=h_{\mathrm{L}, \mathrm{S}}+\left(h_{\mathrm{L}, \mathrm{FI}}-h_{\mathrm{L}, \mathrm{S}}\right)\left(\frac{u_{\mathrm{V}}}{u_{\mathrm{V}, \mathrm{FI}}}\right)^{13}$,
$h_{\mathrm{L}, \mathrm{Fl}}=0.3741 \varepsilon\left(\frac{\eta_{\mathrm{L}} \varrho_{\mathrm{W}}}{\eta_{\mathrm{W}} \varrho_{\mathrm{L}}}\right)^{0.05} \quad \begin{aligned} & \text { for } 0<u_{\mathrm{L}}<200\left[\mathrm{~m}^{3} / \mathrm{m}^{2} \mathrm{~h}\right] \\ & \text { and } \eta_{\mathrm{L}}>10^{-4}[\mathrm{~kg} / \mathrm{ms}] .\end{aligned}$

Above the loading point, Eq. (22) must be modified by an empirical ratio which describes the excess liquid hold-up above this limit up to $h_{\mathrm{L}, \mathrm{Fl}}$; it becomes unity for the range below the loading point (cf. Eq. (29)).
$f(S)=\left(\frac{h_{\mathrm{L}}}{h_{\mathrm{L}, \mathrm{S}}}\right)^{0.3} \exp \left(\frac{\mathrm{Re}_{\mathrm{L}}}{200}\right)$.
A comparison of the pressure drops calculated from Eqs (9) and (10) for dry packed bed and from Eqs (15), (19), (21) and (22) for a packed trickle bed with the constant $C_{\mathrm{P}}$ of Tables 1a and 1 b and the values, determined by experiments up to the loading point, reveals that the mean relative deviation is $9.1 \%$ (see Fig. 7).


Fig. 7. Comparison of pressure drops of dry and trickle beds calculated from Eqs (9) and (15) with experimentally determined values.

Non-steady gas and liquid flow above the loading point makes the precise determination of pressure drop by experiment very difficult. Thus, the scatter range of measured values obtained from these studies is correspondingly larger.

The largest difference occurs in the vicinity of the flood point. Were the review to be extended to include a comparison between the values determined from Eq. (15) and the empirically determined pressure drops up to $90 \%$ of the flood point and if Eq. (27) were adopted for determining the liquid hold-up, the mean relative error would increase to $10.8 \%$.

At very high liquid loads, the film becomes so thick that it coalesces in the narrow parts of the bed. As a result, extensive zones of the fractional free cross-section become filled, and a continuous layer of liquid is thus formed, through which the gas phase rises in the form of bubbles.

If this limiting load, which is referred to as the phase inversion point, is exceeded, the liquid hold-up and the resistance coefficient or the pressure drop in the gas flow increase at a disproportionately high rate. This phenomenon was first described by Elgin and Weiss and by Zenz [7, 8]. It occurs when certain flow parameters reach the value given by Eq. (30). At higher values, Eq. (15) no longer applies.

$$
\begin{equation*}
\frac{L}{V}\left[\frac{\varrho_{\mathrm{V}}}{\varrho_{\mathrm{L}}}\right]^{1 / 2}=0.4 \tag{30}
\end{equation*}
$$

## 4 Conclusions

The equations presented in this paper permit the pressure drop in the gas stream to be predicted mathematically up to the flood point in packed absorption, desorption and rectification columns. They are based on a theoretical model which accounts for the effects of physical, operational and design parameters on the flow of gas and liquid. For the determination of pressure drop, the knowledge of only one constant, specific for the packing, is required, regardless of whether the bed is trickle or dry.

The validity of the derived equations has been confirmed by the fact that the calculated values differ only slightly from the measured ones. The investigated parameters are compiled in Tables 2 and 3.

Received: December 22, 1989 [CET 272]

## Symbols used

| $a$ | $\left[\mathrm{~m}^{2} / \mathrm{m}^{3}\right]$ | total surface area per unit packed volume |
| :--- | :--- | :--- |
| $A_{\mathrm{P}}$ | $\left[\mathrm{m}^{2}\right]$ | total surface area of packing |
| $b$ |  | constant |
| $c$ |  | constant |
| $C_{\mathrm{P}}$ |  | constant |
| $d_{\mathrm{h}}$ | $[\mathrm{m}]$ | hydraulic diameter |
| $d_{\mathrm{P}}$ | $[\mathrm{m}]$ | particle diameter |
| $d_{\mathrm{S}}$ | $[\mathrm{m}]$ | column diameter |
| $f_{\mathrm{w}}$ |  | wetting factor |

Table 2. Pressure and physical properties of investigated systems.

| Systems | $\begin{gathered} p \\ {[\mathrm{mbar}]} \end{gathered}$ | $\begin{gathered} \varrho_{\mathrm{v}} \\ {\left[\mathrm{~kg} / \mathrm{m}^{3}\right]} \end{gathered}$ | $\begin{aligned} & \nu_{\mathrm{v}} \cdot 10^{6} \\ & {\left[\mathrm{~m}^{2} / \mathrm{s}\right]} \end{aligned}$ | $\left[\mathrm{kg} / \mathrm{m}^{3}\right]$ | $\begin{aligned} & \nu_{\mathrm{L}} \cdot 10^{6} \\ & {\left[\mathrm{~m}^{2} / \mathrm{s}\right]} \end{aligned}$ |
| :---: | :---: | :---: | :---: | :---: | :---: |
| Air/water | 1000 | 1.19 | 15.1 | 999 | 1.03 |
| Air/methanol | 1000 | 1.16 | 15.6 | 791 | 0.72 |
| Air/turbine oil | 1000 | 1.13 | 16.1 | 870 | 50.06 |
| Air/machine oil 1 | 1000 | 1.15 | 15.8 | 885 | 58.76 |
| Air/machine oil 2 | 1000 | 1.12 | 15.2 | 890 | 99.03 |
| Air/ethylene glycol | 1000 | 1.17 | 15.4 | 1115 | 16.14 |
| $\mathrm{NH}_{3}$-air/water | 1000 | 1.18 | 15.1 | 999 | 1.03 |
| $\mathrm{NH}_{3}$-air/4\% $\mathrm{H}_{2} \mathrm{SO}_{4}$ in $\mathrm{H}_{2} \mathrm{O}$ | 1000 | 1.18 | 15.2 | 1033 | 1.05 |
| $\mathrm{SO}_{2}$-air/1.78mol. NaOH in $\mathrm{H}_{2} \mathrm{O}$ | 1000 | 1.19 | 15.1 | 1039 | 1.15 |
| $\mathrm{CO}_{2}$-air/water | 1000 | 1.19 | 15.1 | 1000 | 1.00 |
| Methanol/ethanol | 1000 | 1.30 | 8.5 | 738 | 0.52 |
| Ethanol/water | 1000 | 1.29 | 8.4 | 791 | 0.50 |
| Chlorobenzene/ethyl benzene | 33 | 0.14 | 46.0 | 963 | 0.60 |
| Chlorobenzene/cthyl benzene | 67 | 0.27 | 28.9 | 949 | 0.52 |
| Chlorobenzene/cthyl benzene | 133 | 0.51 | 15.9 | 926 | 0.45 |
| Chlorobenzene/ethyl benzene | 266 | 0.96 | 8.8 | 905 | 0.39 |
| Chlorobenzenc/ethyl benzene | 532 | 1.80 | 4.9 | 886 | 0.34 |
| Chlorobenzene/cthyl benzene | 1000 | 3.28 | 2.9 | 866 | 0.30 |
| Toluenc/n-octane | 100 | 0.35 | 20.8 | 839 | 0.52 |
| Toluene $/ n$-octane | 133 | 0.46 | 16.4 | 833 | 0.49 |
| Toluene $/ n$-octane | 266 | 0.90 | 8.5 | 763 | 0.43 |
| Trans-decalin/cis-decalin | 13 | 0.06 | 105.8 | 844 | 1.24 |
| 1.2-Dichloroethane/toluene | 1000 | 3.22 | 2.9 | 924 | 0.38 |
| Ethyl benzene/styrene | 133 | 0.48 | 15.9 | 833 | 0.45 |



Table 3. Capacity range and test facilities.

|  |  |  |  |
| :--- | :---: | :---: | :---: |
| Gas capacity factor | $F_{\mathrm{V}}$ | $\left[\mathrm{m}^{-1 / 2} \mathrm{~s}^{-1} \mathrm{~kg}^{1 / 2}\right]$ | $0.21-5.09$ |
| Liquid load | $u_{1 .} \cdot 10^{3}$ | $\left[\mathrm{~m}^{3} / \mathrm{m}^{2} \mathrm{~s}\right]$ | $0.17-16.7$ |
| Column diameter | $d_{\mathrm{s}}$ | $[\mathrm{m}]$ | $0.15-0.80$ |
| Packed height | $H$ | $[\mathrm{~m}]$ | $0.76-3.95$ |
| Interfacial area | $a$ | $\left[\mathrm{~m}^{2} / \mathrm{m}^{3}\right]$ | $54-380$ |
| Void fraction | $\varepsilon$ | $\left[\mathrm{m}^{3} / \mathrm{m}^{3}\right]$ | $0.66-0.98$ |
| Number of investigated packings |  | 54 |  |
| Number of measurements |  | 3296 |  |


| $F_{\mathrm{V}}$ | $\left[\mathrm{m}^{-1 / 2} \mathrm{~s}^{-1} \mathrm{~kg}^{1 / 2}\right]$ | vapour- or gas capacity factor <br> $g$ |
| :--- | :--- | :--- |
| $\left[\mathrm{m} / \mathrm{s}^{2}\right]$ gravitational acceleration |  |  |
| $h_{\mathrm{L}}$ | $\left[\mathrm{m}^{3} / \mathrm{m}^{3}\right]$ | liquid hold-up |
| $H$ | $[\mathrm{~m}]$ | height |
| $K$ |  | wall factor |
| $L$ | $[\mathrm{~kg} / \mathrm{s}]$ | liquid mass flow rate |
| $N$ | $\left[\mathrm{l} / \mathrm{m}^{3}\right]$ | packing density <br> exponent |
| $n$ |  | pressure drop of trickle packing |
| $\Delta p$ | $[\mathrm{~Pa}]$ | pressure drop of dry packing |
| $\Delta p_{\mathrm{o}}$ | $[\mathrm{Pa}]$ | film thickness |

## Dimensionless numbers

$\operatorname{Re}_{\mathrm{L}}=\frac{u_{\mathrm{L}} \varrho_{\mathrm{L}}}{a \eta_{\mathrm{L}}} \quad$ Reynolds number of liquid $\operatorname{Re}_{v}=\frac{u_{\mathrm{V}} d_{\mathrm{P}}}{(1-\varepsilon) \nu_{\mathrm{V}}} \mathrm{K}$ Reynolds number of gas or vapour

## References

[1] Billet, R., Industrielle Destillation, Veriag Chemie, Weinheim 1973.
[2] Billet, R., Distillation Engineering, Chemical Publishing Company, New York 1979.
[3] Billet, R., I. Chem. Symp. Ser. (1987) No. 104, pp. A171-A182.
[4] Billet, R., Schultes, M., I. Chem. Symp. Ser. (1987) No. 104, pp. A159-A170.
[5] Billet, R., Schultes, M., I. Chem. Symp. Ser. (1987) No. 104, pp. B255-B266.
[6] Brauer, H., Grundlagen der Einphasen- und Mehrphasenströmung, Sauerländer Verlag, Aurau 1971.
[7] Elgin, J.C., Weiss, F.B., Ind. Eng Chem. 31 (1939) pp. 435-445.
[8] Zenz, F.A., Chem. Eng Prog. 8 (1947) pp. 415-428.
[9] Billet, R., Chem. Eng. Technol. 11 (1988) No. 3, pp. 139 - 148.


[^0]:    1) List of symbols at the end of the paper.
